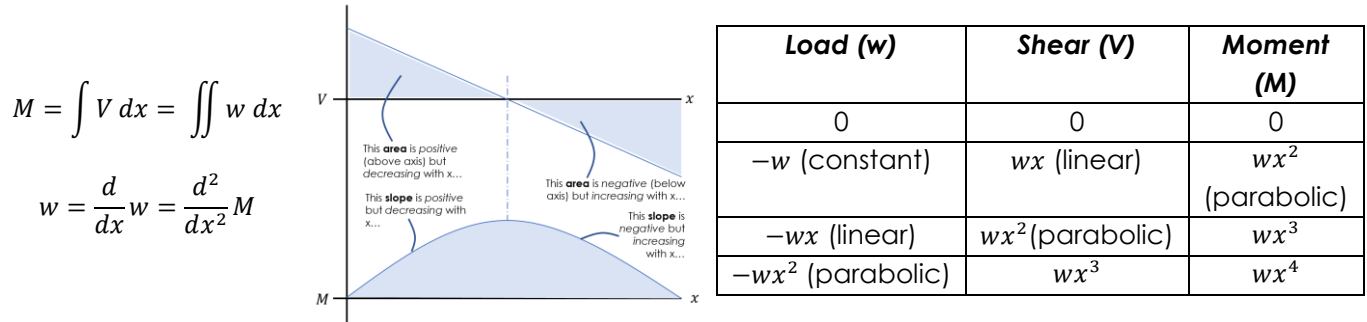


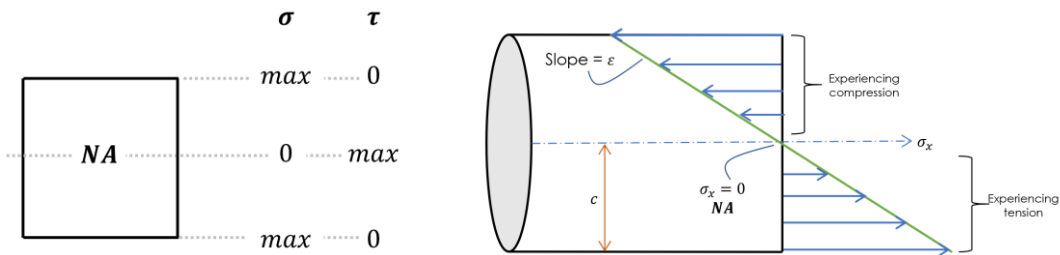
MECHANICS OF MATERIALS: NORMAL & SHEAR STRESS

Normal Stress Caused by Bending:

- Recall shear and moment calculation and graphing techniques:



- The **Neutral Axis** is the axis at which a member in bending experiences no normal stress:



- When bending, we observe how stress varies as a point moves away from the Neutral Axis.

$$\epsilon = -\frac{y}{c} \cdot \epsilon_{max} \quad \text{such that} \quad \sigma_x = E \left(-\frac{y}{c} \right) \cdot \epsilon_{max} \quad \text{and} \quad \sigma_x = -\frac{y}{c} \cdot \sigma_{max}$$

$$I = \int y^2 dA = \frac{Mc}{\sigma_{max}} \quad \text{or} \quad \sigma_x = -\frac{My}{I} \quad \text{for any } y$$

The negative sign is necessary because for a positive y point, the beam experiences compression

- The **Parallel Axis Theorem** is often used for objects that are not strictly rectangular or circular, but rather are comprised of several shapes:

$$I_i = I' + A_i d_i^2$$

Rectangle	$I_{rectangle} = \frac{1}{12} bh^3$
Circle	$I_{circle} = \frac{1}{4} \pi R^4$
Triangle	$I_{triangle} = \frac{1}{36} bh^3$
Semi-circle	$I_{semicircle} = \frac{1}{8} \pi R^4$
b is in the direction parallel (\parallel) to the NA h is in the direction perpendicular (\perp) to the NA	

- **Unsymmetrical bending** due to moments at angles requires the moment vector be decomposed:

The location of the neutral axis is measured by ϕ offset from the positive z-axis.

By knowing the neutral axis experiences no normal stress,

$$\sigma_x = 0 = -\frac{M_z y}{I_z} + \frac{M_y z}{I_y}$$

$$\frac{M_z y}{I_z} = \frac{M_y z}{I_y}$$

$$\frac{y}{z} = \frac{M \sin \theta}{M \cos \theta} \cdot \frac{I_z}{I_y} = \tan \theta \cdot \frac{I_z}{I_y} = \frac{y}{z}$$

Such that

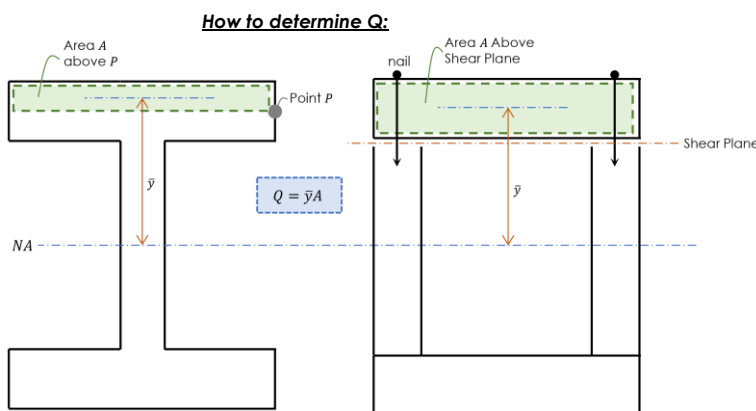
$$\tan \phi = \frac{I_z}{I_y} \cdot \tan \theta$$

$\phi = \tan^{-1} \left(\frac{I_z}{I_y} \cdot \tan \theta \right)$

The ? quadrants depend on the angle of the moment

Shear Stress Caused By Shear Force:

- There are two directions in which shear force acts: longitudinal (along the length of the beam) and transverse (on cut plane)
 1. Horizontal shear (longitudinal): $\Delta H = \frac{VQ}{I} \Delta x$
 2. Transverse shear: $\tau_{avg} = \frac{VQ}{It}$ and $\tau_{max} = 1.5 \cdot \frac{V_{max}}{A}$
- Shear flow (shear force per unit length): $q = \frac{\Delta H}{\Delta x} = \frac{VQ}{I}$



\bar{y}	Distance from NA to the centroid of A
A	Area above the point or above the Shear Plane (opposite side of NA)
I	Moment of inertia of entire object (independent of A or \bar{y})
V	Transverse force applied
t	Thickness of the object at the point observed or Shear Plane